

M. Keller, G. Czilwik *et al.*, Robust temperature change rate actuated siphon valving

1 **Electronic supplementary information (ESI)**

2 **Video S1: Stationary images of a rotating *LabDisk with switch*, recorded by** 3 **the *strobe LabDisk player*.**

4 The video shows the switching of liquid into collection chamber A (Fig. 2, A.1, manuscript)
5 using the same *LabDisk with switch* as in ESI Video S2. The rotational frequency (indicated
6 in the lower left corner) was kept at 15 Hz while cooling down from 60 °C (after holding for
7 60 s) to 30 °C.

8 **Video S2: Stationary images of a rotating *LabDisk with switch*, recorded by** 9 **the *strobe LabDisk player*.**

10 The video shows the switching of liquid into collection chamber B (Fig. 2, B.1-3, manuscript)
11 using the same *LabDisk with switch* as in ESI Video S1. The rotational frequency (indicated
12 in the lower left corner) was first kept at 40 Hz while cooling down from 60 °C (after holding
13 for 60 s) to 30 °C. After 15 s, the rotational frequency was lowered to 9 Hz.

14 **Experimental Details S1: Temperature-dependent dynamic viscosity of air**

15 The dependency of the dynamic viscosity $\eta(T(t))$ of the temperature $T(t)$ (in K) for gases is
16 described by the Sutherland's formula,

$$17 \quad \eta(T(t)) = \eta_0 \frac{T_0 + S}{T(t) + S} \left(\frac{T(t)}{T_0} \right)^{3/2} \quad (\text{S1A})$$

18 where $\eta_0 = 18.27 \cdot 10^{-6}$ Pa s, $T_0 = 291.15$ K, and $S = 120$ K for air¹.

19 **Experimental Details S2: Geometric factor for fluidic resistance**

20 The geometric factor for calculation of the fluidic resistance is calculated as follows² (pages
21 48-51),

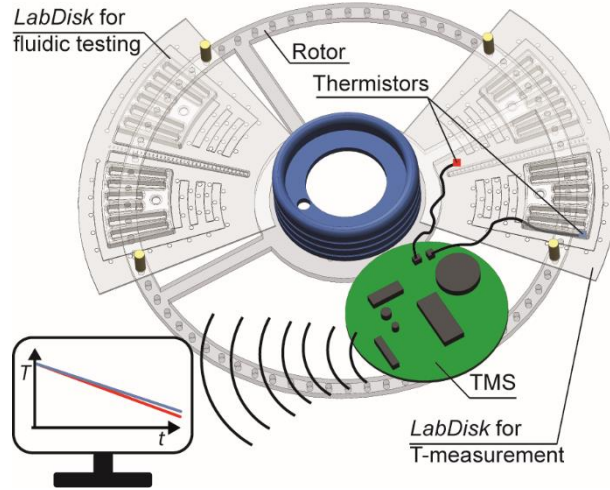
$$22 \quad K = 1 - \sum_{n=1}^{\infty} \frac{1}{(2n-1)^5} \frac{192}{\pi^5} \frac{\min(w_{fl}, d_{fl})}{\max(w_{fl}, d_{fl})} \tanh \left((2n-1) \frac{\pi \max(w_{fl}, d_{fl})}{2 \min(w_{fl}, d_{fl})} \right) \quad (\text{S2A})$$

23 where w_{fl} is the width and d_{fl} is the depth of the venting resistor. $\text{Min}(w_{fl}, d_{fl})$ indicates the
24 smaller value of w_{fl} and d_{fl} . $\text{Max}(w_{fl}, d_{fl})$ indicates the greater value of w_{fl} and d_{fl} .

25 **Experimental Details S3: Model verification**

26 The model was verified using eqn (6) (manuscript), which states that the thermally induced
27 underpressure $\Delta p(t)$ inside the collection chamber may be determined by measuring the

1 capillary and centrifugal pressures in the system (at a known ambient pressure). For this, two
 2 identical *LabDisks for model verification* (segments) together with a rotary temperature
 3 measurement system³ (TMS) were mounted on a rotor for processing inside the *strobe RGQ*
 4 (Fig. S3A).



5
 6 **Fig. S3A** Experimental set-up for verification of the analytical model including a temperature measurement
 7 system (TMS) for measurement under rotation. The TMS is mounted onto a rotor, which holds two identical
 8 *LabDisks for model verification* (segments). One segment serves for temperature measurement and thus includes
 9 thermistors of the TMS (dimension: 0.6 mm x 0.3 mm x 0.3 mm). Data of the TMS is wirelessly transferred to a
 10 computer. Inside the other segment, the fluidic experiment is performed.

11 The inlet chamber of the segment for fluidic testing was filled with 200 μl deionized water
 12 and observed by image acquisition, the other empty segment served for temperature
 13 measurement using the TMS. Due to the thermal masses of the thermistors of the TMS, it was
 14 found most suitable to use a weighted average of the data from the thermistor inside ($T_i(t)$)
 15 and the thermistor outside ($T_o(t)$) of the collection chamber to estimate the actual air
 16 temperature $T(t)$ inside the collection chamber as follows,

$$17 \quad T(t) = 0.6 T_o(t) + 0.4 T_i(t) \quad (\text{S3A}).$$

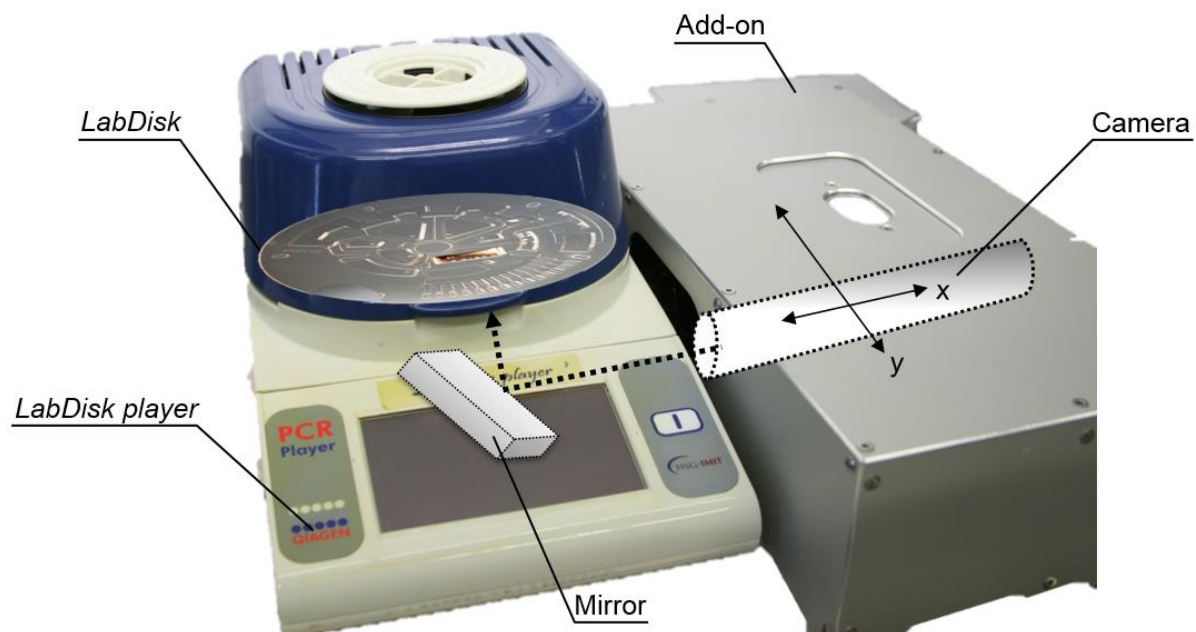
18 Prior to the measurement, the rising channel of the siphon was pre-wetted (by cooling down)
 19 to prevent biasing of the measurement by pinning of the liquid meniscus. Using ImageJ
 20 (National Institutes of Health, Bethesda, MD, USA), the filling heights of the liquid inside the
 21 rising siphon channel ($A = 385 \mu\text{m} \times 385 \mu\text{m}$) and inside the inlet chamber were measured to
 22 determine the centrifugal pressure difference $\Delta p_{\text{cent}}(t)$ (eqn (8), manuscript). Besides, the
 23 contact angle of the liquid inside the siphon channel was observed over time to determine the
 24 capillary pressure $\Delta p_{\text{cap}}(t)$. The corresponding change of contact angle over time $\Theta(t)$
 25 influences on the one hand the measured thermally induced underpressure $\Delta p(t)^{\text{meas}}$ (eqn (6),
 26 manuscript) and on the other hand the underpressure-dependent volume $V(\Delta p(t))$ (eqn (9),

1 manuscript) and thereby the modeled thermally induced underpressure $\Delta p(t)^{\text{mod}}$ (eqn (5),
2 manuscript). Using $\Delta p_{\text{cent}}(t)$ and $\Delta p_{\text{cap}}(t)$, eqn (6) (manuscript) allows to derive a measured
3 thermally induced underpressure $\Delta p(t)^{\text{meas}}$ inside the collection chamber. In addition, the
4 temperature measurement of the TMS was used to calculate a modeled thermally induced
5 underpressure $\Delta p(t)^{\text{mod}}$ inside the collection chamber by numerically solving eqn (5)
6 (manuscript).

7 Both variations were cooled at two different TCR: Since the Rotor-Genie Q/*strobe RGQ* does
8 not allow to define the TCR itself, its RGQ Series Software (version 2.1.0 build 9, QIAGEN)
9 was set to cool down to 35 °C starting from two different temperatures, *i.e.* 60 and 50 °C,
10 which effectively results in different TCR. In addition, even though the siphon channel
11 features a crest on the minimal radial position that can still be observed in the *strobe RGQ*
12 (24.6 mm) to characterize a maximum range of pressure changes, the limitation of the
13 constant 400 RPM of the *strobe RGQ* limits the experimental model verification to the
14 characterization of the influence of the TCR $\dot{T}(t)$ and the fluidic resistance R_{fl} and only up to a
15 maximum of 1.92 kPa thermally induced underpressure. Each experimental configuration is
16 tested three times, which adds up to a set of 12 measurements.

17 **Experimental Details S4: Demonstration of applications**

18 The *LabDisk player* features a processing chamber, where either the *LabDisk with switch* or
19 the *sample-to-answer LabDisk* is placed on a rotor mounted on the rotational axis. Upon
20 closing of the lid of the processing device, a rotational frequency protocol and an air-mediated
21 heating and cooling of the processing chamber can be executed. Thus, the *LabDisk player*
22 uses global heating and cooling of the entire *LabDisk* (structures). As enhancement, the
23 *strobe LabDisk player* allows for observation of the liquid flow inside the rotating respective
24 *LabDisk* (Fig. S4A).



1
2 **Fig. S4A** Experimental set-up of the *strobe LabDisk player*. Liquid flow under rotation is visualized using a
3 camera that acquires picture frames at a defined radial and azimuthal position on the *LabDisk*.

4 **Video S3: Stationary images of a rotating *sample-to-answer LabDisk*,**
5 **recorded by the *strobe LabDisk player*.**

6 The video shows the valving of liquid from the mixing chamber into the aliquoting structure
7 (Fig. 7, C-D, manuscript). The time in ms is indicated in the upper left corner. At a final
8 cooling step from 70 to 50 °C the TCR actuated siphon valve is robustly actuated by allowing
9 valving at low rotational frequency. Liquid is transferred downstream by centrifugal forces
10 into the aliquoting structure for PCR main-amplification.

11 **References**

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