

ESI: Design and Fabrication of a Novel On-Chip Pressure Sensor for Microchannels

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Abstract

This PDF includes:

- Text on calibration setup
- Text on cross-section measurement
- Text on numerical simulation execution
- Text on calibration repeatability test
- Figures S1–S5
- Table I

1 **1. Generation of Constant Pressure Head**

2 The pressure calibration was intended to capture the shapes of fluorescent
3 particle images at various prescribed pressures. The pressure was controlled
4 by a water tank which sustains hydrostatic pressure as shown in Figure 1. By
5 raising or lowering the height of the water tank, the pressure can be changed
6 according to the hydrostatic pressure equation of,

$$p = \rho g(H - h) \tag{1}$$

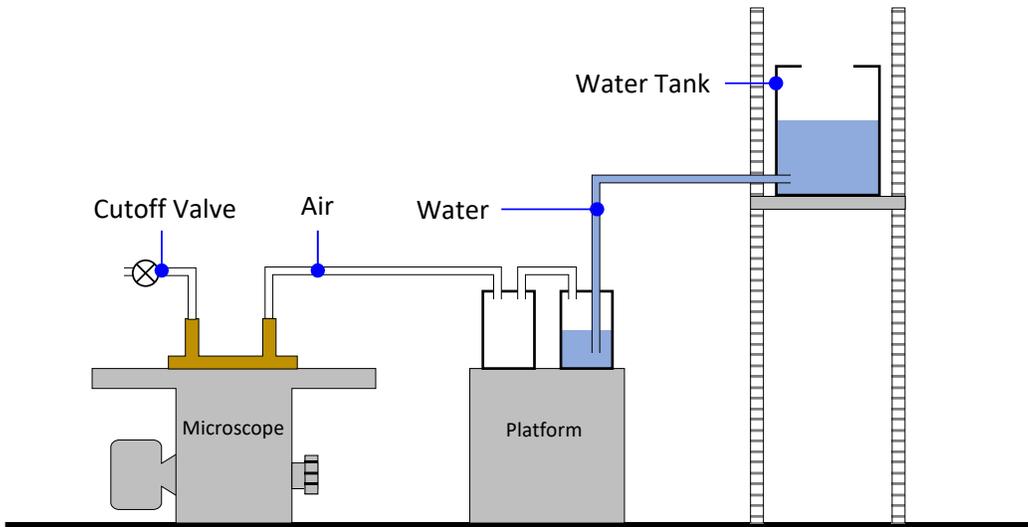


Figure S1: A schematic diagram of the setup used for pressure calibration.

7 where p is the hydrostatic pressure, ρ is the density of the driving fluid, g is
8 the gravitational acceleration, H is the height of the water level in the tank
9 relative to the optical table and h is the height of the water in the glass bottle
10 again relative to the optical table. The water tank and the glass bottle on

11 the right both contains water and the level difference between them creates
12 the net hydrostatic pressure. Here the hydrostatic pressure generated by the
13 air in the tubing is neglected due to its much smaller density. A second glass
14 bottle was used as a buffer chamber to separate from the driving fluid the
15 microchannel to avoid any potential contamination. The pressure calibration
16 was started at zero pressure difference by lowering the tank to the same level
17 of the glass bottle. The pressure was then increased gradually by raising the
18 tank at an increment of 1 cm.

19 **2. Characterization of the Microchannel Cross-Section**

20 It is well known that PDMS microchannels fabricated using soft lithogra-
21 phy often do not have perfectly square or rectangular cross-sections due to
22 the flexible nature of PDMS and other fabrication errors. To ensure accu-
23 rate calculation of numerical and theoretical values for validation purpose,
24 the microchannels used in this study were characterized using an optical
25 profilometer (Profilom3D). Additionally, the cross-section was also directly
26 imaged using a microscope (Olympus IX71) by cutting the microchannel at
27 various locations in the perpendicular direction. Combining the two types of
28 measurements, the actual cross-sectional shape was determined. As shown
29 in Figure 2, the cross-section shape is largely trapezoidal, with curved edges
30 and rounded corners. The height of the microchannel is $\sim 123 \mu\text{m}$. While the
31 top of the microchannel is $\sim 112 \mu\text{m}$ wide, the bottom is only $\sim 96 \mu\text{m}$ wide.
32 This complex geometry of the cross-section coupled with the U-shape design
33 of the microchannel necessitates the use of numerical method to determine
34 the expected values of pressure drop as detailed below.

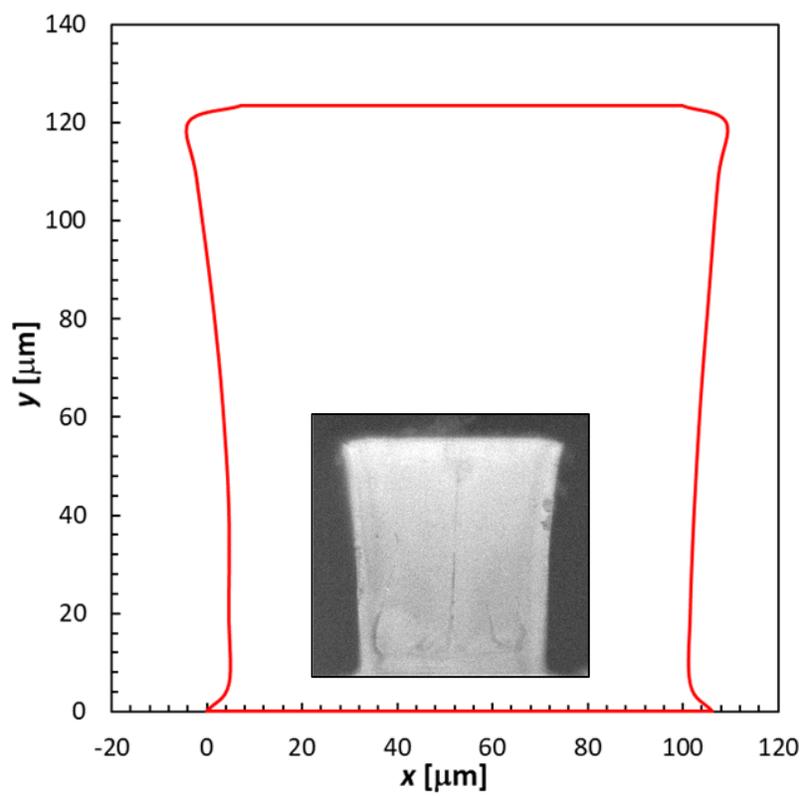


Figure S2: The actual shape of the microchannel cross-section. The inset shows a micro-scope photo of the cross-section.

35 **3. Numerical Simulation in Star-CCM+**

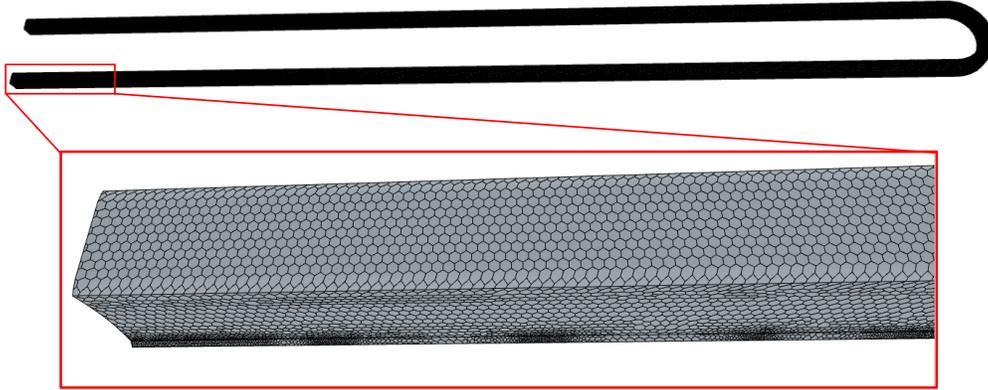


Figure S3: The meshed geometry illustrating the polyhedral meshes.

36 To validate our experimental measurement, the expected pressure drop in the
37 microchannel was numerically solved in Star-CCM+ (v16.04.007). The 3D
38 geometry of the microchannel was based on the original photomask design,
39 reproducing the cross-section characterization in SolidWorks before being im-
40 ported into Star-CCM+. A polyhedral mesh was generated with an average
41 element size of $10\ \mu\text{m}$, resulting a total of 1,121,931 elements. It is worth
42 noting that the base size was selected after a mesh sensitivity analysis was
43 completed at the largest Reynolds number to ensure the wall shear stress
44 was appropriately captured. Figure 3 shows a snapshot of the meshed geom-
45 etry. Given the extremely low Reynolds numbers, the flow was assumed to
46 be incompressible and laminar, with water properties at 23°C (*i.e.*, the mea-
47 sured lab temperature when the experiments were performed). The physical
48 properties of water and air at 23°C are listed in Table 1. A uniform velocity
49 inlet is derived from the desired mass-flow rate and the outlet is a pressure

50 boundary condition set to 0 kPa (gauge). The walls are considered no-slip.
 51 Figure 5 shows a sample pressure field within the microchannel for single-
 52 phase flow of air at 1 ml/min. As expected, the pressure gradually decreases
 53 from the inlet to the outlet. The pressure drop between inlet and outlet at
 54 this condition is 1.02 kPa.

Table 1: Physical properties of water and air at 23°C.

ρ_{water} [kg/m ³]	ρ_{air} [kg/m ³]	μ_{water} [Pa·s]	μ_{air} [Pa·s]
997.5	1.192	9.35×10^{-4}	1.83×10^{-5}

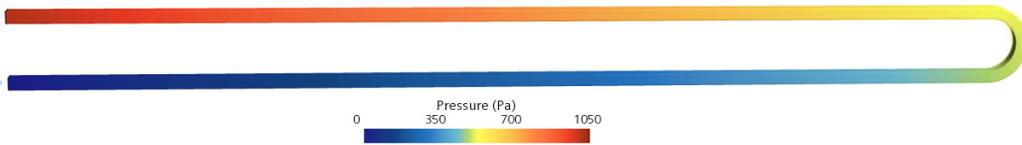


Figure S4: Pressure field within the microchannel obtained from numerical simulation for air flow at 1 ml/min.

55 4. Calibration Repeatability Test

56 To rigorously test the pressure sensor for its robustness and potential hys-
 57 teresis, a test calibration was also performed for 4 consecutive runs using a
 58 separate sensor fabricated in the same way, where the applied pressure was
 59 varied following a pattern of 0 kPa – 2.4 kPa – 0 kPa – 2.4 kPa – 0 kPa at
 60 a step of 0.2 kPa. Essentially, the applied pressure was first increased from
 61 0 kPa to 2.4 kPa at a step of 0.2 kPa (*i.e.*, Run 1), following which the pres-
 62 sure was gradually decreased all the way to 0 kPa (*i.e.*, Run 2). The entire

63 process was then immediately repeated to get Run 3 and Run 4. As shown in
64 Figure S5, the data from all 4 runs agrees very well, with a maximum RMSD
65 of 0.042 kPa (1.75% of the calibrated range) between any two runs, suggest-
66 ing a good repeatability and negligible hysteresis of the pressure sensor in
67 the calibrated range.

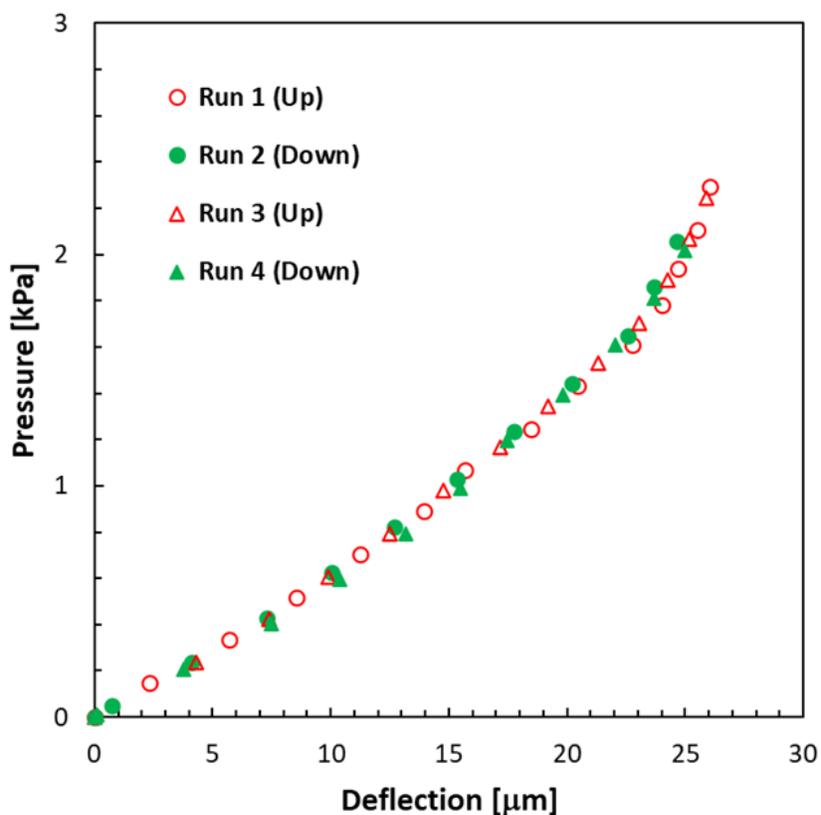


Figure S5: Repeatability test of the pressure calibration. To perform the test, the applied pressure was varied following the pattern of 0 kPa – 2.4 kPa – 0 kPa – 2.4 kPa – 0 kPa at a step of 0.2 kPa. The maximum root mean square deviation between any two runs is 0.042 kPa (1.75%), suggesting a good repeatability and negligible hysteresis of the pressure sensor.