

Supplementary Information

Environmental Assessment of Natural and Fourth Generation Synthetic Refrigerant Blends for Sustainable Cooling in India

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Methodology:

Key performance indicators (KPI):

Evaluation of the initial replacement mixture based on volumetric cooling capacity (VCC), discharge line temperature (DLT) and condenser pressure (Pcond) must replace the refrigerant to ensure system compatibility. Other thermodynamic properties and performance criteria have been included to estimate the pressure ratio of the mixture, including NBP (PR), suction density (ρ_v), cooling effect (RE), capacity cooling per ton (PPTR) and coefficient of performance (COP) (1). These are promising mixtures that minimize NBP, PR and PPTR and maximize RE, ρ_v and COP. In addition, the energy efficiency improvement (η_{II}) was also evaluated with more information about the efficiency of the proposed mixture. These specifications are used to determine the compatibility of tight mixtures in the SS-VCRC cycle used with R744 and the mixtures may have KPI values equal to, lower than, or higher than those predicted by SAFT soft extreme for R744, depending on KPI.

Volumetric cooling capacity (Q_v) and coefficient of performance (COP) are important parameters in the analysis and design of refrigeration systems. Volumetric cooling capacity is the amount of heat absorbed or extracted by the refrigerant per unit volume. It is expressed in units of energy per unit volume (e.g. watts per cubic meter).

$$\Delta Q_v = \dot{m} \cdot \Delta H$$

In which:

- Q_v is the volumetric cooling capacity.
- ' \dot{m} ' is the mass flow rate of the refrigerant (kg/s).
- ΔH is the enthalpy change of the refrigerant during treatment (in J/kg).

Coefficient of performance is a measure of the efficiency of a refrigeration system. It is set differently for heating and cooling mode. The higher the COP, the more efficient is the cooling system. For refrigeration (cooling) systems, COP is calculated by:

$$COP = \frac{Q_c}{W}$$

In which:

- COP is the coefficient of performance.
- Q_c is the cooling capacity of the system "in watts".
- W is the electrical power absorbed by the compressor (in watts).

The disaggregated energy consumption estimates for India's cooling sub-sectors were developed using a bottom-up engineering approach consistent with methods reported by AEEE (2018), the India Cooling Action Plan (ICAP, 2019), and the UNEP-IEA Cooling

Primary data inputs include:

- Equipment stock and sales from AEEE (2018) and BEE Star Label databases;
- Typical cooling capacities and efficiencies (EER/ISEER) from BEE, UNEP–IEA (2020), and manufacturer technical data;
- Annual operating hours derived from ICAP (2019) and climatic degree-day analyses for representative Indian cities;
- Sectoral growth rates and efficiency improvement factors from India Energy Security Scenarios (IESS, NITI Aayog) and AEEE (2018).

Calculation Approach

For each sub-sector i , annual electricity consumption (E_i) was estimated using the standard bottom-up formulation (AEEE, 2018; IEA, 2018):

$$E_i = N_i \times C_i \times \frac{1}{\text{EER}_i} \times H_i$$

where

- N_i = number of installed units (stock),
- C_i = average cooling capacity (kW),
- EER_i = energy efficiency ratio (dimensionless), and
- H_i = equivalent full-load operating hours (h yr^{-1}).

Base-year (2017) stock and efficiency data were compiled from AEEE (2018) and BEE, while H_i values were assigned from ICAP (2019) and Khosla et al. (2021), representing typical residential and commercial duty cycles ($\approx 1,000$ – $1,500 \text{ h yr}^{-1}$). Projections to future years (2027 – 2087) applied sector-specific growth rates and technology-improvement factors from AEEE (2018) and IESS (NITI Aayog), maintaining internal consistency with the CAPEX and TEWI frameworks.

Validation and Sensitivity

Aggregate cooling electricity demand from the above method was cross-validated against national totals reported in ICAP (2019), CEA (2020), and UNEP–IEA (2020), showing agreement within $\pm 10 \%$. Sensitivity analysis ($\pm 20 \%$ variation in H_i and EER_i) indicated that while absolute values shift proportionally, the relative ranking of sub-sectors and comparative TEWI trends remain robust. This confirms that these trends are both realistic and reliable for scenario-based analysis of India's long-term cooling energy and emissions.

Environmental Impact and Economic Analysis

$$TEWI = \{GWP \cdot ((L \cdot m \cdot n) + (m \cdot (1 - \alpha)))\}_{Direct} + \{Ea \cdot \beta \cdot n\}_{Indirect} + AOP$$

$$CAPEX (\$ \cdot y^{-1}) \Sigma C_k = (C_k \times \phi / 3600 \times AOT) \cdot CRF$$

$$CRF = \{i(1 + i)^n / (1 + i)^n - 1\}$$

$$Enviro (\$ \cdot y^{-1}) C_{env} = m_{CO_2eq} \times C_{CO_2} = (\beta \times Ea) \times C_{CO_2}$$

where,

AOT is the Operational hours in a year

AOP is the Atmospheric Oxidation Potential

TEWI is the Total Equivalent Warming Impact (tCO₂eq)

C_{CO₂} Unit cost of CO₂ avoided (US\$.kgCO₂eq⁻¹)

C_{env} is the Penalty cost rate of CO₂ emissions (US\$.y⁻¹)

ΣC_k Capital and maintenance cost rate (US\$.y⁻¹)

φ Maintenance factor

i Annual interest rate

L Annual leakage rate

n System lifetime (y)

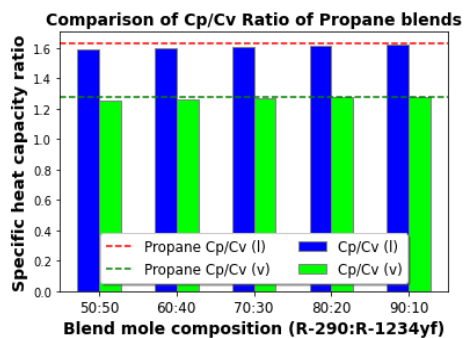
m Refrigerant charge load (kg)

β Indirect emission factor (kgCO₂eq.kWh⁻¹)

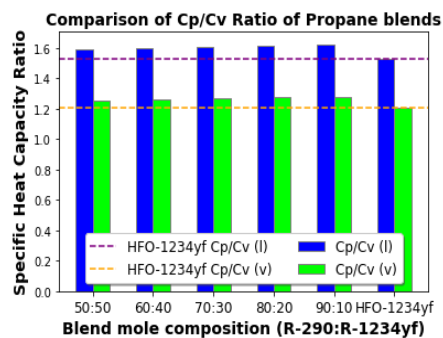
Ea Yearly energy consumption (kWh)

Table S1: Values of various parameters collected from literature

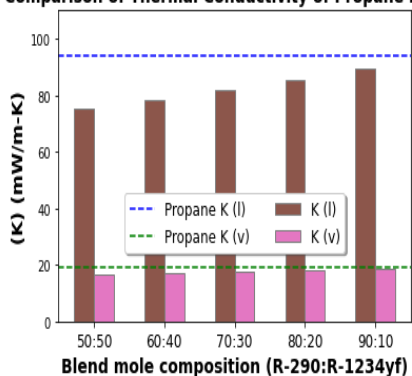
Parameter	Definition	Value	Source
AOT	Operational hours in a year	8760h.y ⁻¹	Alba et al., 2023
C _{CO₂}	Unit cost of CO ₂ avoided	0.087US\$.kgCO ₂ eq ⁻¹)	Adamson et al., 2022 Bamorovat and Kim, 2017
φ	Maintenance factor	1.06	Adamson et al., 2022; Karagoz et al., 2004; Fannou et al., 2015; Heredia et al., 2020
i	Annual interest rate	14%	Adamson et al., 2022; Karagoz et al., 2004; Fannou et al., 2015; Heredia et al., 2020
L	Annual leakage rate	12.5%	Wu et al., 2019
n	System lifetime	15Yr	Gimenez et al., 2022; Abas et al., 2018; Bolaji and Huan, 2013
m	Refrigerant charge load	1.2kg	Alba et al., 2023
β	Indirect emission factor	0.7082kgCO ₂ eq.kWh ⁻¹	Alba et al., 2023
Ea	Yearly energy consumption	485.09kWh/yr	Siddegowda, 2019



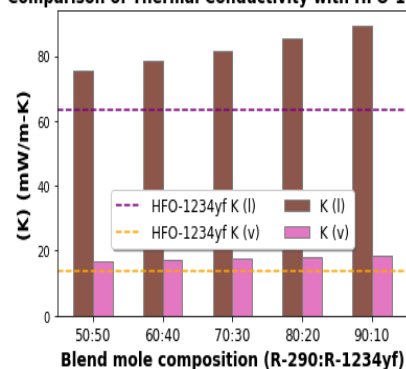
(a)



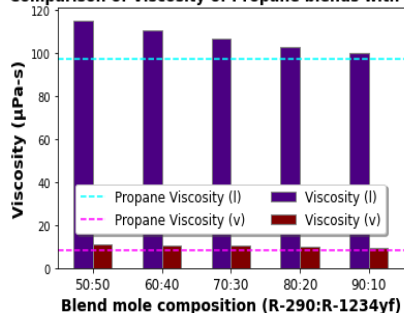
(b)

Comparison of Thermal Conductivity of Propane blends

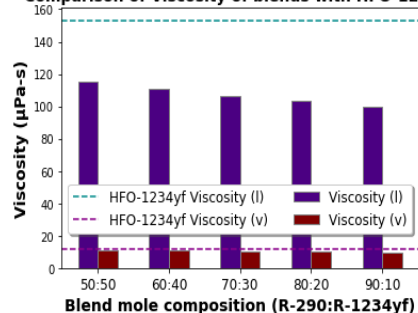
(c)

Comparison of Thermal Conductivity with HFO-1234yf

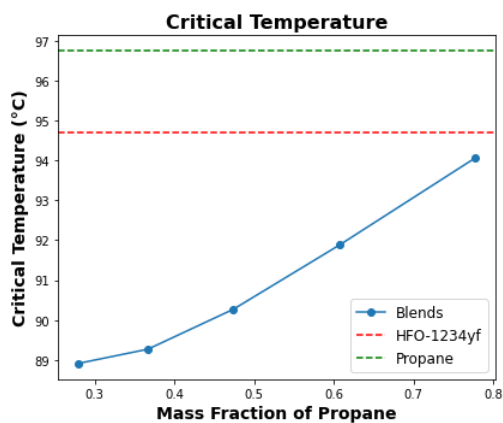
(d)

Comparison of Viscosity of Propane blends with R-290

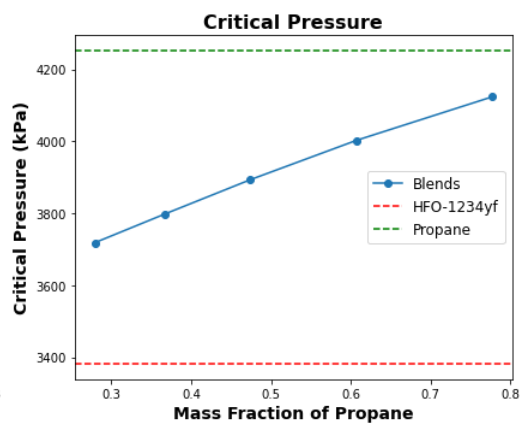
(e)

Comparison of Viscosity of blends with HFO-1234yf

(f)

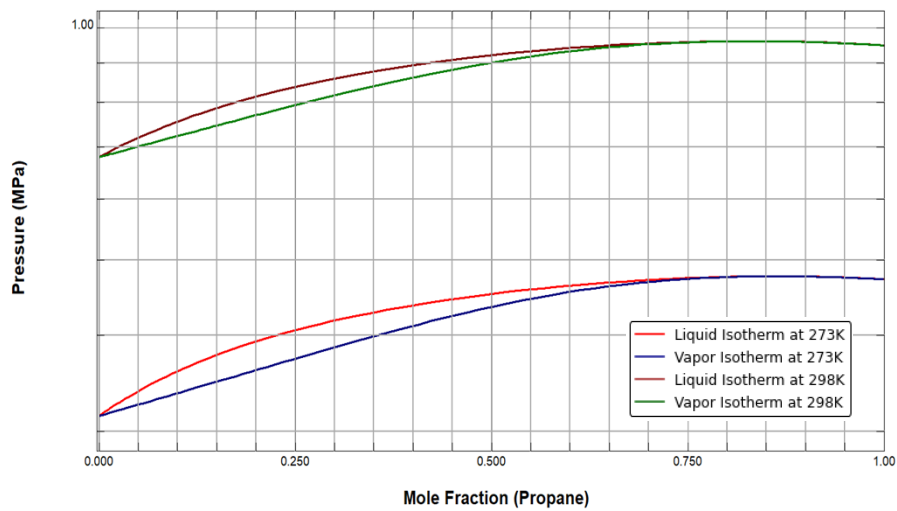


(g)



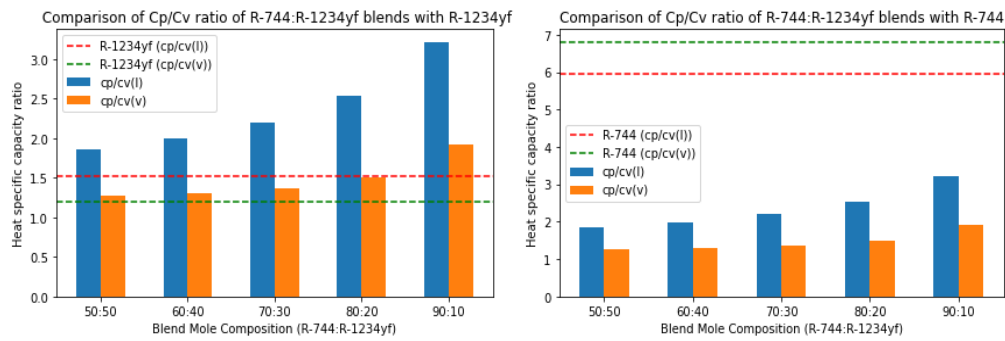
(h)

VLE Diagram of R-290: R-1234yf at 273K and 298K



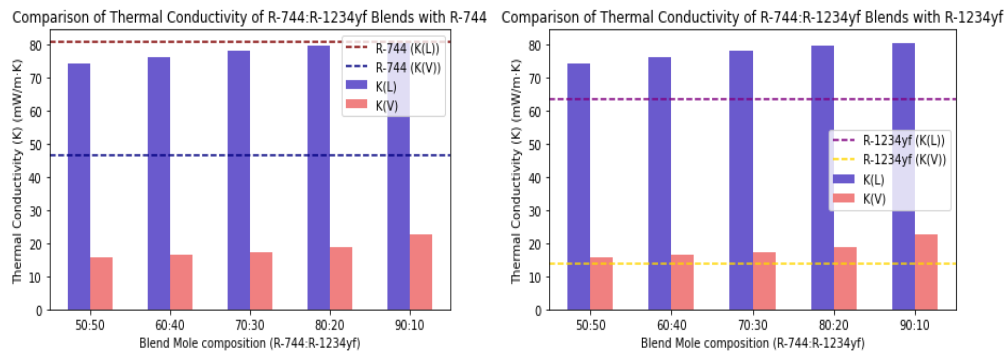
(i)

Figure S1. Thermodynamic properties of the R-290 and R-1234yf blends compared with their individual properties



(a)

(b)



(c)

(d)

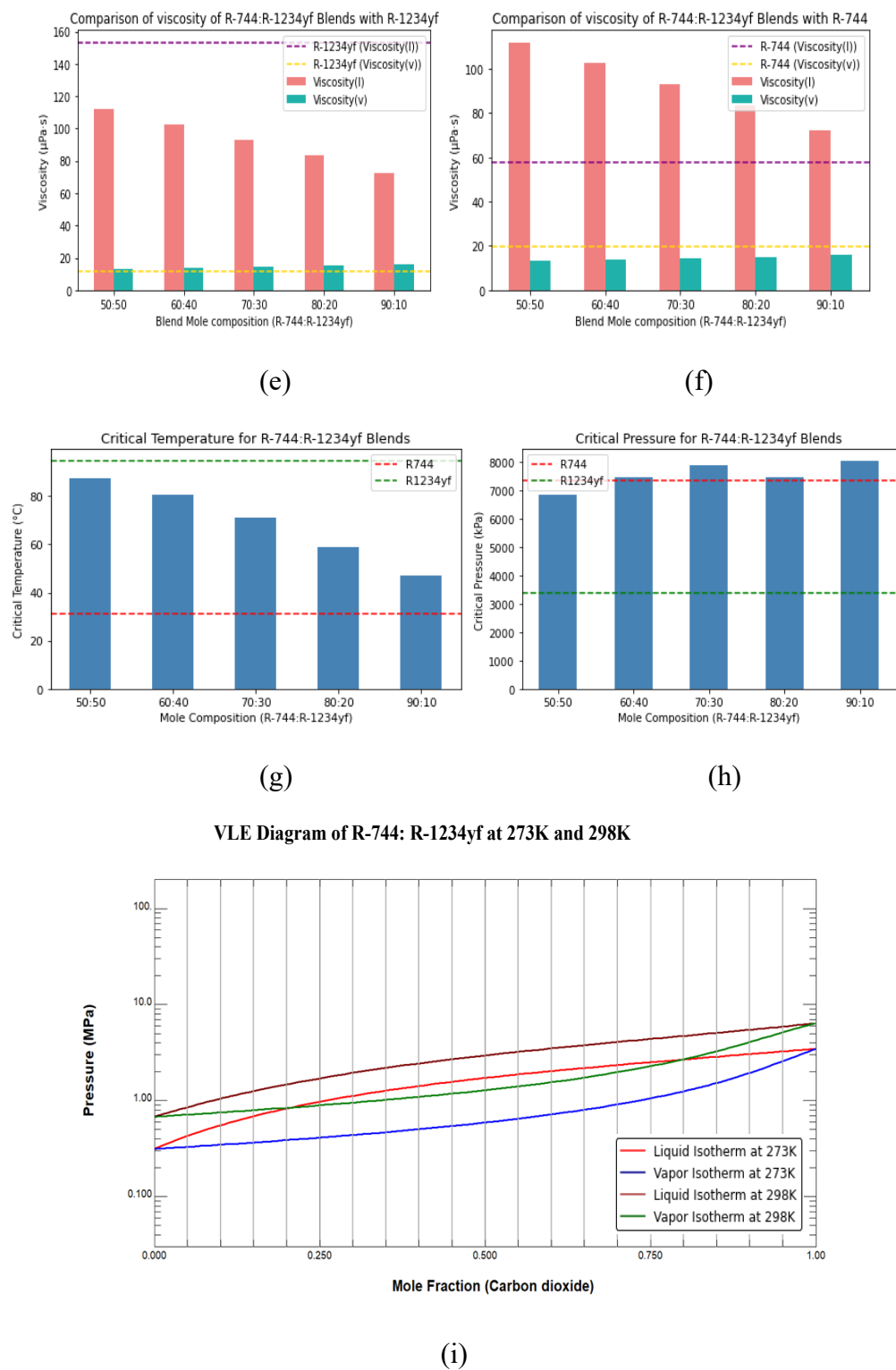


Figure S2. Thermodynamic properties of the R-744 and R-1234yf blends compared with their individual properties

References:

- Albà, C. G., Alkhatib, I. I. I., Llovel, F., & Vega, L. F. (2023). Hunting sustainable refrigerants fulfilling technical, environmental, safety and economic requirements. *Renewable and Sustainable Energy Reviews*, 188, 113806. <https://doi.org/10.1016/j.rser.2023.113806>
- Siddegowda, P. (2019). Experimental investigation of the use of propane for domestic refrigerator with lower displacement compressor. *International Journal of Heat and Technology*, 37(4), 985-990. <https://doi.org/10.18280/ijht.370407>
- Adamson KM, Walmsley TG, Carson JK, Chen Q, Schlosser F, Kong L, et al. Hightemperatureand transcritical heat pump cycles and advancements: areview.*Renew Sustain Energy Rev* 2022;167:112798. <https://doi.org/10.1016/j.rser.2022.112798>.
- Bamorovat Abadi G, Kim KC. Investigation of organic Rankine cycles withzeotropic mixtures as a working fluid: advantages and issues. *Renew Sustain Energy Rev* 2017;73:1000–13. <https://doi.org/10.1016/j.rser.2017.02.020>
- Karagoz S, Yilmaz M, Comakli O, Ozyurt O. R134a and various mixtures of R22/R134a as an alternative to R22 in vapour compression heat pumps.*EnergyConvers Manag* 2004;45:181–96. [https://doi.org/10.1016/S0196-8904\(03\)00144-4](https://doi.org/10.1016/S0196-8904(03)00144-4).
- Fannou J-LC, Rousseau C, Lamarche L, Kajl S. A comparative performance studyof a direct expansion geothermal evaporator using R410A and R407C asrefrigerant alternatives to R22. *Appl Therm Eng* 2015;82:306–17. <https://doi.org/10.1016/j.applthermaleng.2015.02.079>.
- Heredia-Aricapa Y, Belman-Flores JM, Mota-Babiloni A, Serrano-Arellano J,García-Pab'ón JJ. Overview of low GWP mixtures for the replacement of HFC refrigerants: R134a, R404A and R410A. *Int J Refrig* 2020;111:113–23. <https://doi.org/10.1016/j.ijrefrig.2019.11.012>.
- Wu X, Dang C, Xu S, Hihara E. State of the art on the flammability ofhydrofluoroolefin (HFO) refrigerants. *Int J Refrig* 2019;108:209–23. <https://doi.org/10.1016/j.ijrefrig.2019.08.025>
- Gim'enez-Prades P, Navarro-Esbrí J, Arpagaus C, Fern'andez-Moreno A, Mota-Babiloni A. Novel molecules as working fluids for refrigeration, heat pump andorganic Rankine cycle systems. *Renew Sustain Energy Rev* 2022;167. <https://doi.org/10.1016/j.rser.2022.112549>.
- Abas N, Kalair AR, Khan N, Haider A, Saleem Z, Saleem MS. Natural and syntheticrefrigerants, global warming: a review. *Renew Sustain Energy Rev* 2018;90:557–69. <https://doi.org/10.1016/j.rser.2018.03.099>.
- Bolaji BO, Huan Z. Ozone depletion and global warming: case for the use ofnatural refrigerant - a review. *Renew Sustain Energy Rev* 2013;18:49–54. <https://doi.org/10.1016/j.rser.2012.10.008>.

Sanguri K, Ganguly K, Pandey A. Modelling the barriers to low global warming potential refrigerants adoption in developing countries: a case of Indian refrigeration industry. *J Clean Prod* 2021;280:124357. <https://doi.org/10.1016/j.jclepro.2020.124357>.

ASHRAE Standard 34 (2019). *Designation and Safety Classification of Refrigerants*.

Polonara F., Kuijpers L.J.M., Peixoto R.A., Arnaldo M.B. (2017). *Int. J. Heat Technol.*, 35(S1), S1–S8.

Bell I.H., Domanski P.A., McLinden M.O., Linteris G.T. (2019). *Int. J. Refrigeration*, 104, 484–495.

Kivevele T. (2022). *Automotive Experiences*, 5(1), 75–89.

Smith C. et al. (2021). *IPCC AR6 – The Earth's Energy Budget, Table 7.SM.6*.

Lemmon E.W., Bell I.H., Huber M.L., & McLinden M.O. (2018). NIST Standard Reference Database 23: REFPROP v10.0. National Institute of Standards and Technology.

UNEP (2016). Decision XXVII/4 Task Force Report – Further Information on Alternatives to Ozone-Depleting Substances.